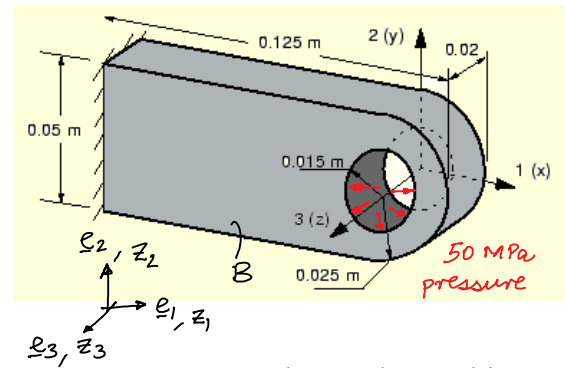


## Chapter 5: Boundary value problems in solid mechanics

Recall governing equations:

$$\begin{aligned}
 \text{GDE} \left\{ \begin{array}{l} \text{div } \underline{\underline{S}} + \underline{b} = \underline{0} \quad \text{--- (EQ)} \quad \forall \underline{x} \in \phi(B) \\ \underline{S} = \underline{S}^T \quad \forall \underline{x} \in \phi(B) \\ \underline{\underline{E}} = \frac{1}{2} (\nabla \underline{u} + \nabla \underline{u}^T) \quad \text{--- (SD)} \quad \forall \underline{x} \in \phi(B) \\ \underline{S} = \lambda \text{tr}(\underline{\underline{E}}) \underline{I} + 2\mu \underline{\underline{E}} \quad \text{--- (SS)} \quad \forall \underline{x} \in \phi(B) \end{array} \right. \\
 \text{BC} \left\{ \begin{array}{l} \underline{u} = \underline{u}_D \quad \forall \underline{x} \in \phi(A_D) \\ \underline{t}(\underline{m}) = \underline{S} \underline{n} = \underline{h}_N \quad \forall \underline{x} \in \phi(A_N) \end{array} \right.
 \end{aligned}$$



Boundary Value Problem

Example: Pure bending of a prismatic cantilever beam:  
(pages 250-255, Timoshenko & Goodier)

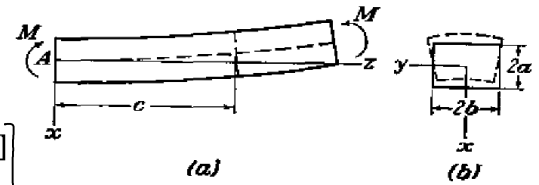


FIG. 141.

Map:  $\underline{x} = \underline{\xi} + \underline{u}(\underline{\xi})$

$$\underline{x} \sim \begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix} \sim \begin{Bmatrix} z_1 \rightarrow x \\ z_2 \rightarrow y \\ z_3 \rightarrow z \end{Bmatrix} \quad \underline{u} \sim \begin{cases} u = -\frac{1}{2R} [z^2 + v(x^2 - y^2)] \\ v = -\frac{vxy}{R} \\ w = \frac{xz}{R} \end{cases}$$

Note: For a cross section at  $z = c$ :

$$x_3 = c + w = c + \frac{cx}{R}$$

These kinds of maps are solved for using techniques for PDEs. But work only for simple geometries and loadings.

Note: For the lateral surfaces of the beam:

$$\begin{aligned}
 x_2 &= \pm b + v = \pm b \left(1 - \frac{vx}{R}\right) \\
 x_1 &= \pm a + u = \pm a - \frac{1}{2R} [c^2 + v(a^2 - y^2)]
 \end{aligned}$$

Strains:  $\underline{\underline{E}} = \frac{1}{2} (\nabla \underline{u} + \nabla \underline{u}^T)$

$$\underline{\underline{E}} \sim \frac{1}{2} \left\{ \begin{bmatrix} -\frac{2x}{R} & \frac{2y}{R} & -\frac{z}{R} \\ -\frac{2y}{R} & -\frac{2x}{R} & 0 \\ -\frac{z}{R} & 0 & \frac{x}{R} \end{bmatrix} + \begin{bmatrix} -\frac{2x}{R} & -\frac{2y}{R} & \frac{z}{R} \\ \frac{2y}{R} & -\frac{2x}{R} & 0 \\ -\frac{z}{R} & 0 & \frac{x}{R} \end{bmatrix} \right\} \sim \begin{bmatrix} -\frac{2x}{R} & 0 & 0 \\ 0 & -\frac{2x}{R} & 0 \\ 0 & 0 & \frac{x}{R} \end{bmatrix}$$

Stress:

$$\underline{\underline{S}} = \frac{C \gamma}{(1+\nu)(1-2\nu)} \text{tr}(\underline{\underline{E}}) \underline{I} + \frac{C}{(1+\nu)} \underline{\underline{E}} \sim \frac{C}{(1+\nu)} \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & (1+\nu) \frac{x}{R} \end{bmatrix}$$

$$\Rightarrow S_{33} = \frac{Cx}{R}$$

{ Recall:  $\frac{\sigma}{y} = \frac{E}{R} = \frac{M}{I}$  }

Note:  $\text{div } \underline{\underline{S}} + \underline{b} = 0$  ← assumed "0" + Check BCs.

Read Example 27 from the textbook.  
(See. 101 from Timoshenko & Goodier)

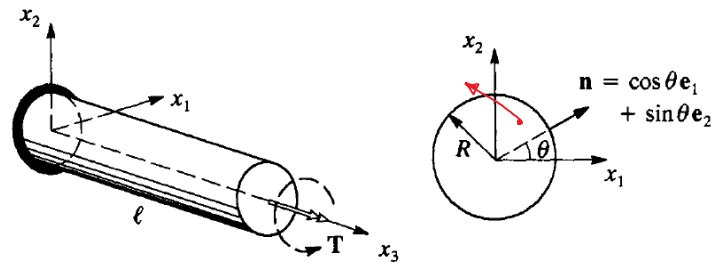


Figure 63 Pure torsion of a circular shaft

Map:  $\underline{x} = \underline{z} + \underline{u}(\underline{z})$   
 $\underline{x} = \underline{z} - \beta z_2 z_3 \underline{e}_1 + \beta z_1 z_3 \underline{e}_2$

Strains:

$$\underline{\epsilon} \sim \frac{1}{2} \beta \begin{bmatrix} 0 & -z_3 & -z_2 \\ -z_3 & 0 & z_1 \\ -z_2 & z_1 & 0 \end{bmatrix}$$

$\Rightarrow$  Stress:  $\underline{\sigma} = \lambda \text{tr}(\underline{\epsilon}) \underline{\mathbb{I}} + 2\mu \underline{\epsilon} = \mu \beta \begin{bmatrix} 0 & 0 & -z_2 \\ 0 & 0 & z_1 \\ -z_2 & z_1 & 0 \end{bmatrix}$

Verify  $\text{div } \underline{\sigma} + \underline{b} = \underline{0}$  + check BCs.

These systems of PDEs, cannot be solved analytically, in general.  
We use numerical methods and use simplifying assumptions.

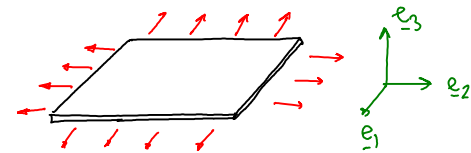
Simplifying Assumptions:

- Axisymmetric problems
  - Plane Stress / Plane Strain
  - Beam Theory
  - Plate / Shell Theory
- } Structural Mechanics

Example: 2D Plane Problems

• Plane Stress

$$\sigma_{33} = 0 \quad ; \quad \sigma_{13} = \sigma_{31} = 0 \quad ; \quad \sigma_{23} = \sigma_{32} = 0$$



Stress-strain relationship:

$$\begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix} = \frac{E}{(1-\nu^2)} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1-\nu}{2} \end{bmatrix} \begin{Bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ 2\epsilon_{xy} \end{Bmatrix}$$

ie  $\underline{\sigma} = \underline{D}_{\text{ps}} \underline{\epsilon}$   
and  $\epsilon_{zz} = -\frac{\nu}{E} (\sigma_{xx} + \sigma_{yy})$

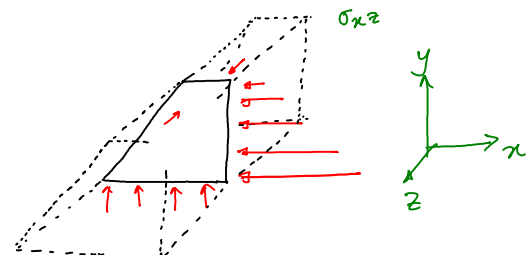
• Plane Strain

$$\epsilon_{33} = 0 \quad ; \quad \epsilon_{13} = \epsilon_{31} = 0 \quad ; \quad \epsilon_{23} = \epsilon_{32} = 0$$

Stress-strain relationship:

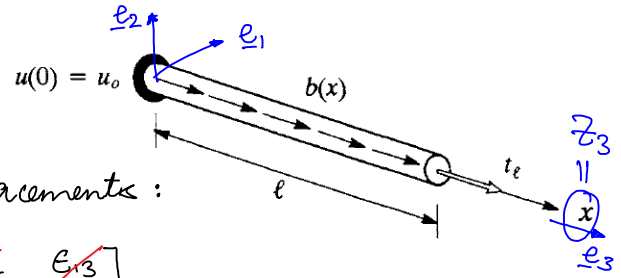
$$\begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix} = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} (1-\nu) & \nu & 0 \\ \nu & (1-\nu) & 0 \\ 0 & 0 & \frac{1-2\nu}{2} \end{bmatrix} \begin{Bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ 2\epsilon_{xy} \end{Bmatrix}$$

ie  $\underline{\sigma} = \underline{D}_{\text{pe}} \underline{\epsilon}$



### One-dimensional (1D) little BVP

To understand how to obtain numerical solutions to complicated 2D/3D problems in general, let's first study some 1-D problems where we can usually obtain exact solutions.



Assuming small strains and displacements:

$$\underline{u} \sim \begin{bmatrix} u_1 \\ u_2 \\ u_3 \end{bmatrix} \rightarrow u(x) \quad \underline{\epsilon} \sim \begin{bmatrix} \epsilon_{11} & \epsilon_{12} & \epsilon_{13} \\ \epsilon_{21} & \epsilon_{22} & \epsilon_{23} \\ \epsilon_{31} & \epsilon_{32} & \epsilon_{33} \end{bmatrix} \rightarrow \frac{\partial u_3}{\partial z_3} = u'(x) = \frac{du}{dx}$$

$$\Rightarrow \underline{\underline{s}} = \frac{C \gamma}{(1+\gamma)(1-2\gamma)} \text{tr}(\underline{\epsilon}) \underline{\underline{I}} + \frac{C}{(1+\gamma)} \underline{\epsilon} \Rightarrow \frac{C}{(1+\gamma)} \begin{bmatrix} \gamma \epsilon_{33} + \epsilon_{11} & 0 & 0 \\ 0 & \gamma \epsilon_{33} + \epsilon_{22} & 0 \\ (1+\gamma) \epsilon_{33} & 0 & \gamma \epsilon_{33} + \epsilon_{33} \end{bmatrix} \Rightarrow \begin{matrix} s_{33} = C \epsilon_{33} \\ \sigma = C \epsilon \end{matrix}$$

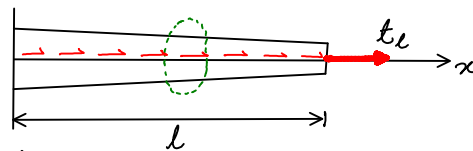
Thus Governing equations:

GDE	{	$\text{div } \underline{\underline{s}} + \underline{b} = \underline{0} \Rightarrow$	$\frac{d\sigma}{dx} + b(x) = 0 \Rightarrow \sigma' + b = 0 \quad \forall x \in (0, l)$
		$\underline{\epsilon} = \frac{1}{2} (\nabla \underline{u} + \nabla \underline{u}^T) \Rightarrow$	$\epsilon = u' = du/dx \quad \forall x \in (0, l)$
		$\underline{\underline{s}} = \lambda \text{tr}(\underline{\epsilon}) \underline{\underline{I}} + 2\mu \underline{\epsilon} \Rightarrow$	$\sigma = C \epsilon \quad \forall x \in (0, l)$
BCs.	{	$\underline{t}_{(n)} = \underline{\underline{s}} \underline{n} = \underline{t}_n \Rightarrow$	$t_l = \sigma(l) (+) \quad \text{at } x=l$
		$\underline{u} = \underline{u}_D \Rightarrow$	$u(0) = u_0 \quad \text{at } x=0$

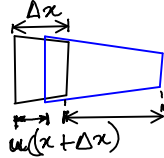
Substituting for  $\epsilon(x)$  and  $\sigma(x)$ :

$$\begin{cases} (Cu')' + b = 0 & \forall x \in (0, l) \\ Cu'(l) (+) = t_l & \text{at } x=l \\ u(0) = u_0 & \text{at } x=0 \end{cases}$$

Alternative derivation:



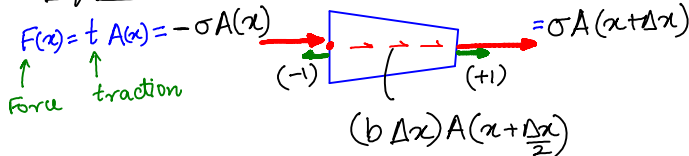
Kinematics:



Strain (1D)

$$\epsilon(x) = \lim_{\Delta x \rightarrow 0} \frac{u(x+\Delta x) - u(x)}{\Delta x} \Rightarrow \epsilon(x) = \frac{du}{dx} = u'$$

Equilibrium



Material:  $\sigma = C \epsilon$

$$\sum F_x = 0 \Rightarrow \sigma A(x+\Delta x) - \sigma A(x) + b A \Delta x = 0$$

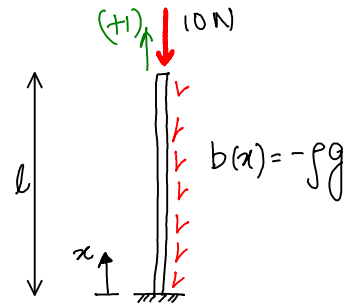
$$\Rightarrow \lim_{\Delta x \rightarrow 0} \frac{\sigma A(x+\Delta x) - \sigma A(x)}{\Delta x} + b A = 0 \Rightarrow \frac{d(\sigma A)}{dx} + b A = 0$$

If  $A(x) = \text{constant} \Rightarrow$

$$\sigma' + b = 0$$

Example. Column under self weight:

- $A(x) = \text{constant} = 1$  (say)
- $\sigma(x) = C \epsilon(x)$  : constant "C"
- Density :  $\rho$
- BCs :  $u(0) = 0$  on  $\Gamma_D$  ( $x=0$ )  
 $\sigma(l)(+) = t_L = -10$  on  $\Gamma_L$  ( $x=l$ )



Find  $\sigma(x)$  ;  $u(x)$ .

For stresses:

$$\sigma' + b = 0$$

$$\Rightarrow \frac{d\sigma}{dx} = -b(x) = +\rho g$$

$$\Rightarrow \sigma(x) = \rho g x + c_1$$

$$t_L = \sigma(l) = \rho g l + c_1$$

BC:

$$\sigma(l)(+) = t_L = -10$$

$$\Rightarrow c_1 = t_L - \rho g l$$

$$\Rightarrow \boxed{\sigma(x) = \rho g x - \rho g l + t_L}$$

Alternatively:

$$\sigma(l) - \sigma(x) = \int_x^l \rho g dx = \rho g (l-x) \quad (\text{Integrating } \int_x^l \bullet)$$

$$\Rightarrow \boxed{\sigma(x) = \rho g x - \rho g l + t_L} \quad (t_L = \sigma(l)(+))$$

For displacement:

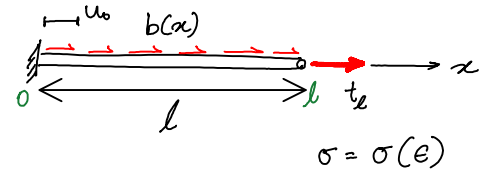
$$C u' = \sigma(x) = \rho g x - \rho g l + t_L$$

$$\Rightarrow \boxed{u(x) = \frac{1}{C} \left[ \rho g \frac{x^2}{2} - \rho g l x + t_L x \right]} + c_2 \quad \begin{matrix} u_0 = 0 \\ \text{BC} \\ (u(0) = u_0 = 0) \end{matrix}$$

Strong Forms and Weak forms

Consider the 1-D problem:

Find  $\sigma(x), u(x)$ :



$$\textcircled{S} \begin{cases} \text{GDE} \left[ \begin{array}{l} \sigma' + b = 0 \\ u(0) = u_0 \end{array} \right. & \forall x \in (0, l) \\ \text{BC} \left[ \begin{array}{l} \sigma(l)(+) = t_2 \end{array} \right. & \text{on } \Gamma_b \\ & \text{on } \Gamma_N \end{cases}$$

Residual:  
 $g(x) = \sigma' + b$

This is called the Strong Form  $\textcircled{S}$  of the governing differential equation (GDE).

Weak forms

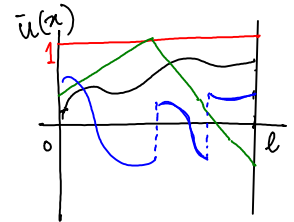
Define  $G(\sigma, \bar{u}) \equiv \ominus \left[ \int_0^l \bar{u}(x) g(x) \cdot dx + \bar{u}(0)(t_0 + \sigma(0)) + \bar{u}(l)(t_2 - \sigma(l)) \right]$

$G(\sigma, \bar{u})$ : A scalar functional (function of functions)

Example:  $G\left(\frac{1}{2}x^2, x\right) = - \int_0^l x \cdot x dx - 0(t_0 + 0) - l(t_2 - l) = -\frac{l^3}{3} - l(t_2 - l)$   
 Input functions  $g = \sigma' + b = (x + b)$   
 scalar  $+\frac{bl^2}{2}$

Weak form of the Problem Statement.

If for some  $\sigma(x), t_0$   $G(\sigma, \bar{u}) = 0 \quad \forall \bar{u} \in V(0, l)$  *for all Function space*



Then  $\Rightarrow \begin{cases} g = \sigma' + b = 0 & \forall x \text{ in } (0, l) \\ \sigma(l)(+) = t_2 & \text{at } x=l \\ \sigma(0)(-) = t_0 & \text{at } x=0 \end{cases}$

This relies on the fundamental theorem of Calculus of Variations:

If  $G(\sigma, \bar{u}) = \ominus \left[ \int_0^l \bar{u}(x) \overbrace{g(x)}^{\sigma' + b} dx - \bar{u}(0) \overbrace{a_0}^{t_0 + \sigma(0)} - \bar{u}(l) \overbrace{a_l}^{t_2 - \sigma(l)} \right] = 0 \quad \forall \bar{u}(x) \in V(0, l)$

Then  $\Rightarrow g(x) = 0 \quad \text{and} \quad a_0 = 0 \quad \text{and} \quad a_l = 0$

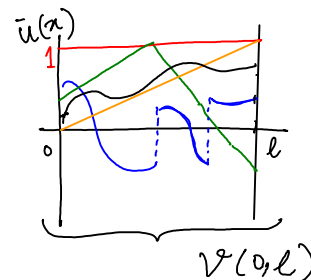
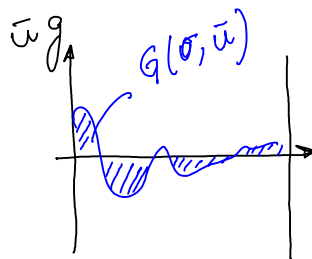
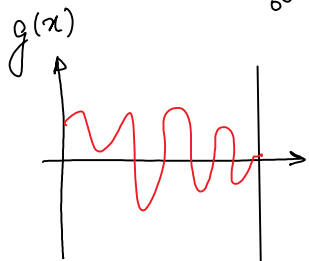
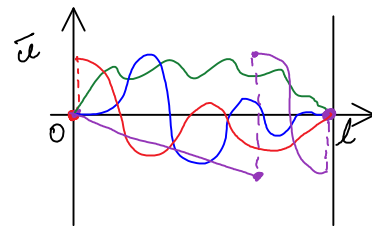
Proof of Fundamental theorem of calculus of variations

First, let  $\bar{u}(0) = \bar{u}(l) = 0$

There exists  $\bar{u}(x) = g(x) \in V(0,l)$

So if  $G(\sigma, \bar{u}) = -\int_0^l \bar{u} g dx$

$$= -\int_0^l g^2 dx = 0 \Rightarrow g = 0 \Rightarrow \sigma' + b = 0 \quad \forall x \in (0,l)$$



Now in addition if  $\bar{u}(0), \bar{u}(l) \neq 0 \Rightarrow$

$$\text{If } G(\sigma, \bar{u}) = -\int_0^l \bar{u} g dx - \bar{u}(0)(t_0 + \sigma(0)) - \bar{u}(l)(t_l - \sigma(l)) = 0 \quad \forall \bar{u} \in V(0,l)$$

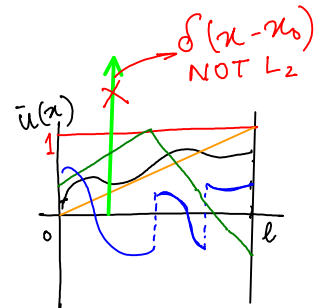
$$\Rightarrow \quad t_0 + \sigma(0) = 0 \quad ; \quad t_l - \sigma(l) = 0$$

Restriction on the choice of Function spaces  $V(0,l)$

Fundamental Theorem of Calculus of variations restricts  $\bar{u} \in V(0,l)$ :

$\bar{u}$  must be square integrable

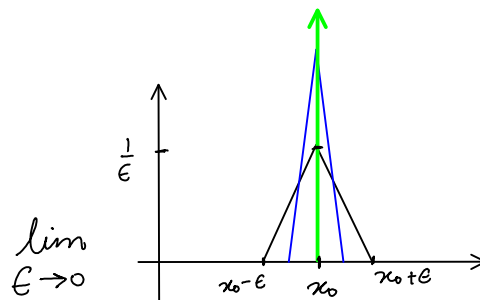
ie  $\underbrace{\int_0^l (\bar{u})^2 dx}_{L_2\text{-norm of } \bar{u}(x)} \text{ must exist (be finite)}$  (L: Lebesgue)



Note: Dirac-delta  $\delta(x-x_0)$  is not  $L_2$ .

$$\int_{-\infty}^{\infty} \delta(x-x_0) dx = 1$$

$$\int_{-\infty}^{\infty} [\delta(x-x_0)]^2 dx = \frac{2}{3\epsilon} \rightarrow \infty \text{ (as } \epsilon \rightarrow 0)$$



Possible choices for function spaces:

- Square integrable:  $L_2$  (or  $H^0$ )
- Sq. Int. upto 1<sup>st</sup> derivative:  $H^1$
- Sq. Int. upto  $m^{\text{th}}$  derivative:  $H^m$
- Continuous functions:  $C^0$
- Continuous upto 1<sup>st</sup> der.:  $C^1$
- Continuous upto  $m^{\text{th}}$  der.:  $C^m$

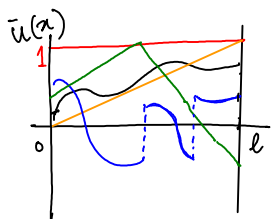
$$C^\infty \subset C^m \subset H^m \subset \dots \subset C^1 \subset H^1 \subset C^0 \subset (H^0 = L_2)$$

subset

$C^\infty$   
smooth  
(polynomial, Trigs, Exp)

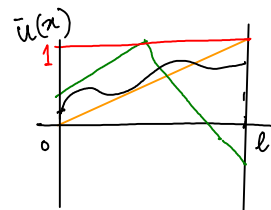
Examples of function spaces:

$L_2 / H^0$ :



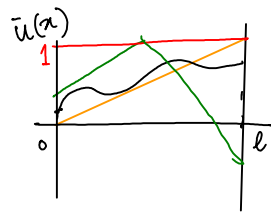
(discontinuous functions are possible)

$C^0$



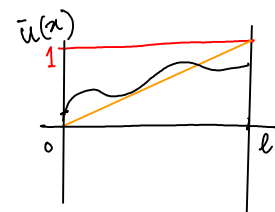
continuous functions only

$H^1$ :



(first derivatives square integrable)

$C^1$



(first derivative continuous)

Example of Approximate solution to Weak form:

Recall exact solution:

$$\sigma(x) = \rho g x - \rho g l + t_L$$

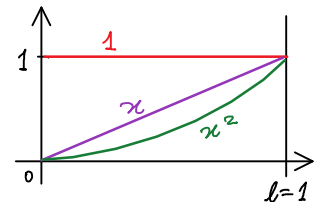
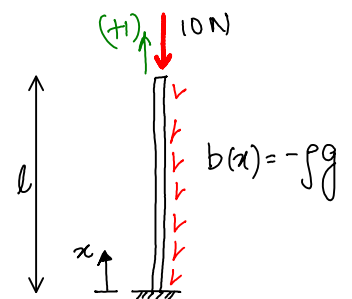
Ⓜ Find  $\sigma, t_0$  such that  
 $G(\sigma, \bar{u}) = 0 \quad \forall \bar{u} \in V(0,1)$

Let  $\sigma = a_0 + a_1 x + a_2 x^2$   $\left\{ \begin{array}{l} a_i: \text{Unknown} \\ \bar{a}_i: \text{Arbitrary} \end{array} \right.$   
 and  $\bar{u} = \bar{a}_0 1 + \bar{a}_1 x + \bar{a}_2 x^2$

For  $G(\sigma, u) = 0$

$$\Rightarrow - \int_0^1 (\bar{a}_i \bar{h}_i(x)) \cdot (a_j h_j(x)) dx - \bar{u}(0)(t_0 + \sigma(0)) - \bar{u}(l)(t_L - \sigma(l)) = 0$$

$$\Rightarrow \bar{a} [K \underline{a} - \underline{f}] = 0 \quad \forall \bar{a}_i \in \mathbb{R}$$



Weak Form: Method of weighted Residuals (MWR):

MWR (W)  $G(\sigma, \bar{u}) \equiv \ominus \int_0^l \bar{u}(x) g(x) \cdot dx - \bar{u}(0)(t_0 + \sigma(0)) + \bar{u}(l)(\sigma(l) - t_2)$

$$G(\sigma, \bar{u}) = \ominus \int_0^l \bar{u}(\sigma' + b) dx - \bar{u}(0)(\sigma(0) + t_0) + \bar{u}(l)(\sigma(l) - t_2)$$

$$= - \int_0^l d(\bar{u}\sigma) + \int_0^l \bar{u}'\sigma dx - \int_0^l \bar{u}b dx \quad \left| \begin{array}{l} (\bar{u}\sigma)' = \bar{u}'\sigma + \bar{u}\sigma' \\ \Rightarrow \ominus \bar{u}\sigma' = \ominus \frac{d(\bar{u}\sigma)}{dx} \oplus \bar{u}'\sigma \end{array} \right.$$

(Integration by parts)

$$= - [\bar{u}\sigma]_0^l + \int_0^l \bar{u}'\sigma dx - \int_0^l \bar{u}b dx$$

$$= \cancel{\bar{u}(0)\sigma(0)} - \cancel{\bar{u}(l)\sigma(l)} + \int_0^l \bar{u}'\sigma dx - \int_0^l \bar{u}b dx$$

Unknown reactions  $\rightarrow$  Applied traction  $\checkmark$

Weak Form: Principle of Virtual Work (PVW):

PVW (W)  $\Rightarrow G(\sigma, \bar{u}) = \underbrace{\int_0^l (\bar{u}'\sigma) dx}_{W_I} - \underbrace{\int_0^l \bar{u}b dx - t_2 \bar{u}(l) - t_0 \bar{u}(0)}_{-W_E}$

Virtual Strain  $\bar{u}' = \bar{\epsilon}$

Internal Virtual Work

External Virtual Work.

Substituting the material model:  $\sigma = C u'$

Weak form problem statements:

MWR (W)

Find  $u(x) \in H^2(0, l)$  and  $t_0$  such that  $u(0) = u_0$   
 $G(u, \bar{u}) = 0 \quad \forall \quad \bar{u}(x) \in L_2(0, l) = H^0(0, l)$   
 where  $G(u, \bar{u}) \equiv \int_0^l \bar{u}((Cu')' + b) dx - \bar{u}(0)(t_0 + \sigma(0)) - \bar{u}(l)(t_2 - \sigma(l))$

OR

PVW (W)

Find  $u(x) \in H^1(0, l)$  and  $t_0$  such that  $u(0) = u_0$   
 $G(u, \bar{u}) = 0 \quad \forall \quad \bar{u}(x) \in \cancel{L_2(0, l)} \neq H^1(0, l)$   
 where  $G(u, \bar{u}) \equiv \int_0^l \bar{u}' Cu' dx - \int_0^l \bar{u}b dx - \bar{u}(0) t_0 - \bar{u}(l) t_2$



## Boundary Conditions

In the weak form of the problem statement:

MWR  $(\bar{w})$ : Find  $t_0$  and  $u(x) \in H^2(0,l)$  such that  $u(0) = u_0$  (ESSENTIAL BC  $(\Gamma_D: x=0)$ )  
 and  $G(\sigma, \bar{u}) = 0 \quad \forall \quad \bar{u}(x) \in H^0(0,l)$   
 where  $G(\sigma, \bar{u}) \equiv -\int_0^l \bar{u}(\sigma' + b) dx - \bar{u}(0)(t_0 + \sigma(0)) - \bar{u}(l)(t_2 - \sigma(l))$  (NATURAL BC  $(\Gamma_N: x=l)$ )

OR

PVW  $(\bar{w})$ : Find  $t_0$  and  $u(x) \in H^1(0,l)$  such that  $u(0) = u_0$  (ESSENTIAL BC)  
 and  $G(\sigma, \bar{u}) = 0 \quad \forall \quad \bar{u}(x) \in H^1(0,l)$   
 where  $G(\sigma, \bar{u}) \equiv \underbrace{\int_0^l \bar{u}' \sigma dx}_{W_I} - \left[ \underbrace{\int_0^l \bar{u} b dx + \bar{u}(0) t_0 + \bar{u}(l) t_2}_{W_E} \right]$

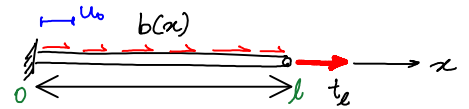
Unknowns to solve for:  $\sigma(x)$  &  $t_0$  (or  $u(x)$  &  $t_0$  where  $\sigma = \sigma(u')$ )

But  $t_0 = \sigma(0)(-1)$ , so we can eliminate  $t_0$ .

By further restricting  $\bar{u} = 0$  on  $\Gamma_D$  (even if actual  $u_0 \neq 0$ )  
 HOMOGENEOUS ESSENTIAL BC.

Define the spaces:  
 (for virtual displacement)  $H_E^0(0,l) \equiv H^0(0,l)$  and  $\bar{u} = 0$  on  $\Gamma_D$   
 $H_E^1(0,l) \equiv H^1(0,l)$  and  $\bar{u} = 0$  on  $\Gamma_D$

Thus the final weak forms are:



MWR  $(\bar{w})$ : Find  ~~$t_0$  and~~  $u(x) \in H^2(0,l)$  such that  $u(0) = u_0$  (EBC  $(\Gamma_D: x=0)$ )  
 and  $G(\sigma, \bar{u}) = 0 \quad \forall \quad \bar{u}(x) \in H_E^0(0,l)$  (space for virtual disp.)  
 where  $G(\sigma, \bar{u}) \equiv -\int_0^l \bar{u}(\sigma' + b) dx - \bar{u}(l)(t_2 - \sigma(l))$  (HEBC) (NBC  $(\Gamma_N: x=l)$ )

OR

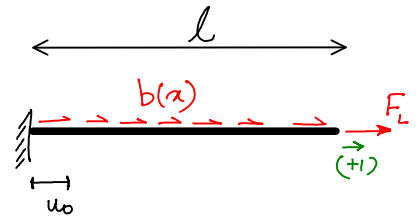
PVW  $(\bar{w})$ : Find  ~~$t_0$  and~~  $u(x) \in H^1(0,l)$  such that  $u(0) = u_0$  (EBC)  
 and  $G(\sigma, \bar{u}) = 0 \quad \forall \quad \bar{u}(x) \in H_E^1(0,l)$  (space for virtual disp.)  
 where  $G(\sigma, \bar{u}) \equiv \underbrace{\int_0^l \bar{u}' \sigma dx}_{W_I} - \left[ \underbrace{\int_0^l \bar{u} b dx + \bar{u}(l) t_2}_{W_E} \right]$  (HEBC)

Note:  $(S) \iff (W)$

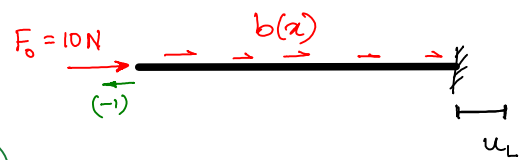
Examples of Boundary Conditions & Approximation Function Spaces

⑤ Strong form  $(Cu')' + b = 0 \quad \forall x \in (0, l)$  (assume  $A=1$ )  
 (+ some BCs)

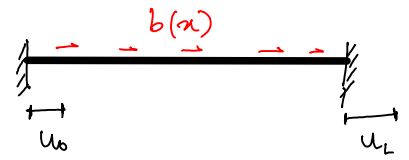
①  $u(x=0) = u_0$  EBC  $\Rightarrow \bar{u}(0) = 0$  (HEBC)  
 $\sigma(l)(+1) = t_L = \frac{F_L}{A}$  NBC



②  $\sigma(0)(-1) = t_0 = \frac{F_0}{A}$  NBC  
 $u(x=l) = u_L$  EBC  $\Rightarrow \bar{u}(l) = 0$  (HEBC)



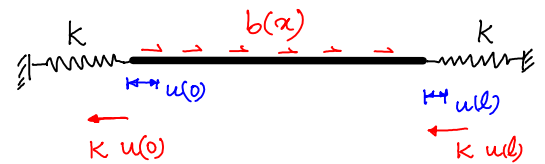
③  $u(x=0) = u_0$  EBC  $\Rightarrow \bar{u}(0) = 0$  (HEBC)  
 $u(x=l) = u_L$  EBC  $\Rightarrow \bar{u}(l) = 0$  (HEBC)



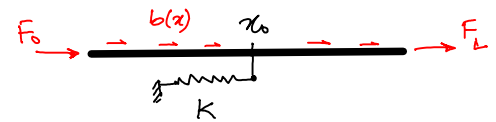
④  $\sigma(0)(-1) = t_0 = \frac{F_0}{A}$  NBC  
 $\sigma(l)(+1) = t_L = \frac{F_L}{A} = -\frac{K u(l)}{A}$  Mixed BC



⑤  $\sigma(0)(-1) = t_0 = -\frac{K u(0)}{A}$  (Mixed)  
 $\sigma(l)(+1) = t_L = -\frac{K u(l)}{A}$  (Mixed)



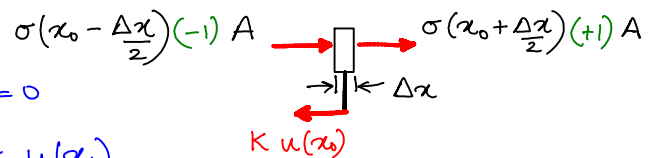
⑥  $\sigma(0)(-1) = t_0 = \frac{F_0}{A}$   
 $\sigma(l)(+1) = t_L = \frac{F_L}{A}$  } NBCs



Jump condition

$$\sigma(x_0)|_+ - \sigma(x_0)|_- - K u(x_0) = 0$$

$$\Rightarrow \llbracket \sigma(x_0) \rrbracket = K u(x_0)$$

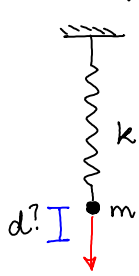


## Variational (Energy-based) methods

{Ref. Hjeltnstad ch 9

In contrast with weighted residuals, sometimes it is possible to derive the weak form from Variational (Energy) Principles.

### Example



Equilibrium (MWR/PVW)

(PBD)



$$\Rightarrow kd = mg$$

$$\Rightarrow d = \frac{mg}{k}$$

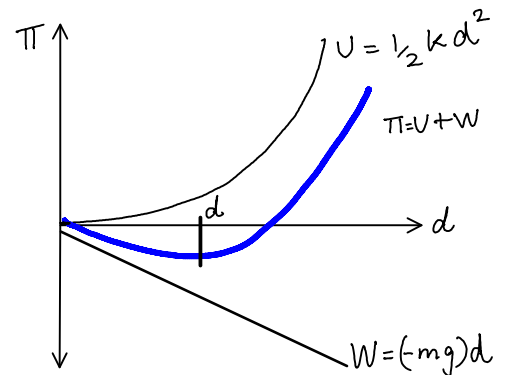
Minimization of Potential Energy

$$\Pi = U + W$$

$$\Pi = \frac{1}{2} k d^2 + (-mgd)$$

$$\text{Minimum} \Rightarrow \frac{\partial \Pi}{\partial d} = 0 \Rightarrow kd - mg = 0$$

$$\Rightarrow kd = mg \text{ (Equilibrium)}$$



Now lets consider:

$$Cu'' + b = 0 \quad \text{for } x \in (0, l)$$

$$u(0) = u_0 \quad ; \quad Cu'(l) = t_2$$

Total Potential Energy

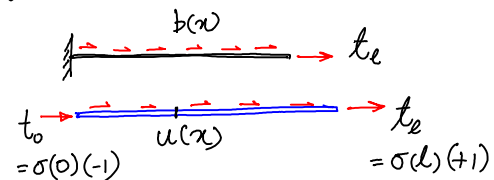
$$\Pi = U + W$$

↓ Potential due to conservative body forces & external tractions

Recall Strain energy

$$U(u) = \int_0^l \underbrace{\frac{1}{2} \sigma \epsilon}_{\psi(\epsilon)} dx = \int_0^l \frac{1}{2} C(u')^2 dx$$

$$W(u) = \int_0^l -ub dx \quad -u(l)t_2 - u(0)t_0$$



$$\text{Thus } \Pi(u) = \int_0^l \left[ \frac{1}{2} C(u')^2 - ub \right] dx - u(l)t_2 - u(0)t_0$$

Note: Real displacement only

To minimize Potential Energy  $\Pi(u)$  wrt  $u(x)$  we need: Directional Derivative (Gateaux Derivative)

$$D \Pi(u) \cdot \bar{u} = \left[ \frac{d}{d\epsilon} \left( \Pi(u + \epsilon \bar{u}) \right) \right]_{\epsilon=0} = 0$$

Energy weak form:

$$\begin{aligned}
 \mathbb{D} \Pi(u) \cdot \bar{u} &= \left[ \frac{d}{d\epsilon} [\Pi(u + \epsilon \bar{u})] \right]_{\epsilon=0} = \frac{d}{d\epsilon} \left[ \int_0^l \frac{1}{2} c (u' + \epsilon \bar{u}')^2 dx - \int_0^l (u + \epsilon \bar{u}) b dx \right. \\
 &\quad \left. - (u(0) + \epsilon \bar{u}(0)) t_0 - (u(l) + \epsilon \bar{u}(l)) t_l \right] \\
 &= \frac{d}{d\epsilon} \left[ \int_0^l \frac{1}{2} c (u'^2 + 2\epsilon u' \bar{u}' + \epsilon^2 \bar{u}'^2) dx - \int_0^l (u + \epsilon \bar{u}) b dx \right. \\
 &\quad \left. - [u(0) + \epsilon \bar{u}(0)] t_0 - [u(l) + \epsilon \bar{u}(l)] t_l \right]
 \end{aligned}$$

i.e.

$$= \underbrace{\int_0^l c u' \bar{u}' dx}_{W_I} - \underbrace{\int_0^l \bar{u} b dx + \bar{u}(l) t_l + \bar{u}(0) t_0}_{W_E} = G(u, \bar{u}) !$$

$\mathbb{D} \Pi(u) \cdot \bar{u} = G(u, \bar{u})$

$\textcircled{E} \Rightarrow \textcircled{W}$

$\textcircled{W} \quad \boxed{\text{PVW}}$

Integrate by parts (in reverse - to unbalance the derivatives):

$$\begin{aligned}
 &= - \int_0^l \bar{u} C u'' dx + [\bar{u} C u']_0^l - \int_0^l \bar{u} b dx - \bar{u}(l) t_l - u(0) t_0 \\
 &= - \int_0^l \bar{u} \underbrace{(C u'' + b)}_{\textcircled{S}} dx + \bar{u}(l) \underbrace{[C u'(l) - t_l]}_{\text{Natural BC @ } l} - \bar{u}(0) \underbrace{[C u'(0) + t_0]}_{\text{Nat BC @ } 0}
 \end{aligned}$$

Now if

$$\mathbb{D} \Pi(u) \cdot \bar{u} = G(u, \bar{u}) = 0 \quad \text{for all: } \forall \bar{u}(x) \in H_0^1(0, l)$$

then

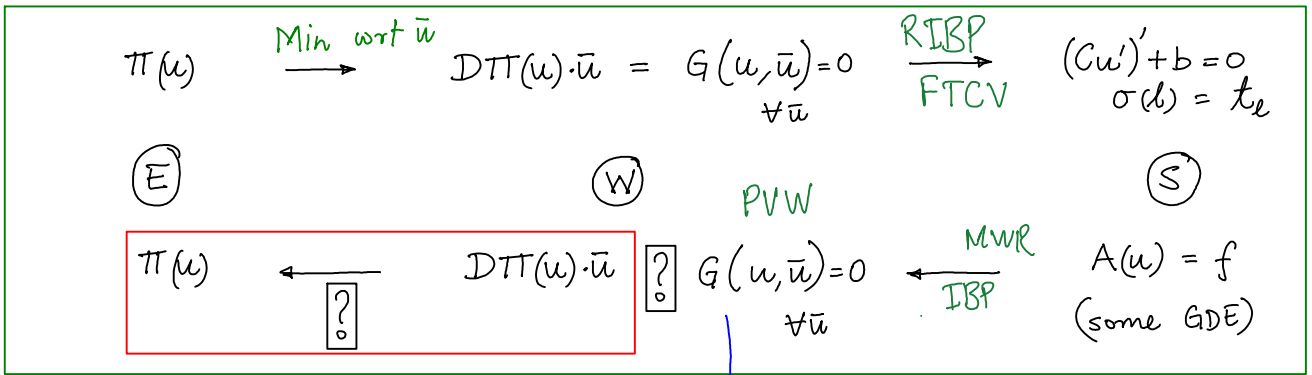
$$\textcircled{S} \left[ \begin{array}{l} C u'' + b = 0 \quad \text{at all points } x \text{ in } (0, l) \\ C u'(l) = \sigma(l) = t_l \quad \text{at } x = l \end{array} \right.$$

i.e. we get the governing differential equation (GDE) back from the variational principle.

In general, if you have "some"  $\boxed{\Pi(u)}$   
i.e. some "energy" functional

then, the governing differential equation corresponding to  $\Pi(u)$  is called its Euler equation (or Euler-Lagrange equation)  $\textcircled{S}$

Vainberg's Theorem: Existence of an Energy form (Variational form)



Existence of a Variational Principle is decided with the help of the Vainberg's Theorem:

$G^h(d, \bar{d})$  Approximate solutions

Vainberg's Theorem

Given a functional  $G(u, \bar{u})$  {i.e PVW (W) or GDE strong form (S)}

If •  $G(u, \bar{u})$  is linear in the second argument:

ie  $G(u, (\alpha \bar{u}_1 + \beta \bar{u}_2)) = \alpha G(u, \bar{u}_1) + \beta G(u, \bar{u}_2)$

• Directional derivative is symmetric in the second argument:

ie  $DG(u, \bar{u}_1) \cdot \bar{u}_2 = DG(u, \bar{u}_2) \cdot \bar{u}_1$

Then  $\left\{ \text{where } DG(u, \bar{u}_1) \cdot \bar{u}_2 \equiv \left[ \frac{d}{d\epsilon} G(u + \epsilon \bar{u}_2, \bar{u}_1) \right] \Big|_{\epsilon=0} \right\}$

$\Pi(u) = \int_0^1 G(tu, u) dt + c$  such that  $DTT(u) \cdot \bar{u} = G(u, \bar{u})$

Example:

Consider the Weak form for the 1-D problem:-

(W)

$$G(u, \bar{u}) = \int_0^l \bar{u}' C u' dx - \int_0^l \bar{u} b dx - \bar{u}(l) t_2$$



Vainberg's Th:

• Linearity of  $\bar{u}$  ✓

• Symmetric in  $\bar{u}$  ✓  $D G(u, \bar{u}_1) \cdot \bar{u}_2 \stackrel{?}{=} D G(u, \bar{u}_2) \cdot \bar{u}_1$

LHS

$$D G(u, \bar{u}_1) \cdot \bar{u}_2 = \frac{d}{d\epsilon} \left[ \int_0^l \bar{u}_1' C (u + \epsilon \bar{u}_2)' dx - \int_0^l \bar{u}_1 b dx - \bar{u}_1(l) t_2 \right]_{\epsilon=0}$$

$$= \int_0^l \bar{u}_1' C \bar{u}_2' dx \quad (*)$$

RHS

$$D G(u, \bar{u}_2) \cdot \bar{u}_1 = \frac{d}{d\epsilon} \left[ \int_0^l \bar{u}_2' C (u + \epsilon \bar{u}_1)' dx - \int_0^l \bar{u}_2 b dx - \bar{u}_2(l) t_2 \right]_{\epsilon=0}$$

$$= \int_0^l \bar{u}_2' C \bar{u}_1' dx \quad (*)$$

$\Rightarrow \Pi(u)$  exists!

$$\int_0^1 t dt = \left[ \frac{t^2}{2} \right]_0^1 = \frac{1}{2}$$

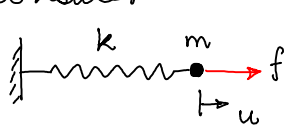
$$\Pi(u) = \int_0^1 G(tu, u) dt + c$$

$$= \int_0^1 \left[ \int_0^l u' C (tu)' dx - \int_0^l u b dx - u(l) t_2 \right] dt$$

$$\Rightarrow \Pi(u) = \int_0^l \frac{1}{2} C (u')^2 dx - \int_0^l u b dx - u(l) t_2 \quad (E)$$

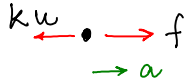
## Hamilton's Principle (for Dynamics)

Consider



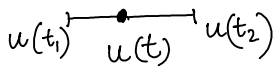
Equation of motion: (Dynamic Equilibrium  $F = ma$ )

FBD:

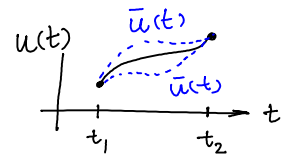


$$m \ddot{u} + k u - f = 0 \quad \forall t$$

Using Energy principles (Variational methods):



$$K(\dot{u}) = \frac{1}{2} m \dot{u}^2 \quad ; \quad \Pi(u) = \frac{1}{2} k u^2 - f \cdot u$$



Define: Lagrangian

$$\mathcal{L}(u, \dot{u}) = K(\dot{u}) - \Pi(u)$$

Hamiltonian

$$H(u, p) = \underbrace{K(p)}_{\frac{1}{2} \frac{p^2}{m}} + \Pi(u)$$

Hamilton's Principle:

$$\mathcal{D} \left[ \int_{t_1}^{t_2} \mathcal{L}(u, \dot{u}) dt \right] \cdot \bar{u} = 0 \quad \forall \bar{u}(t)$$

Action

$$\Rightarrow \frac{d}{d\epsilon} \left[ \int_{t_1}^{t_2} \left[ \frac{1}{2} m (\dot{u} + \epsilon \dot{\bar{u}})^2 - \frac{1}{2} k (u + \epsilon \bar{u})^2 + f \cdot (u + \epsilon \bar{u}) \right] dt \right]_{\epsilon=0} = 0$$

$$\Rightarrow \int_{t_1}^{t_2} (m \dot{u} \dot{\bar{u}} - k u \bar{u} + f \bar{u}) dt = 0 \quad \forall \bar{u}(t)$$

(Integrate by parts in t)

$$\Rightarrow \int_{t_1}^{t_2} (-\bar{u} m \ddot{u} - k u \bar{u} + f \bar{u}) dt + [m \dot{u} \bar{u}]_{t_1}^{t_2} = 0 \quad \forall \bar{u}(t)$$

( $\bar{u}(t_1) = \bar{u}(t_2) = 0$ )

$$\Rightarrow \boxed{m \ddot{u} + k u - f = 0}$$

This is the Euler-Lagrange equation corresponding to the Lagrangian above.

Using the same Lagrangian  $\mathcal{L}(u, \dot{u}) = K(\dot{u}) - \Pi(u)$

For 1-D problem:

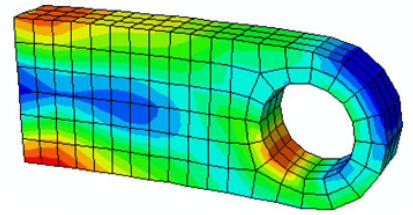
$$\sigma' + b = f \ddot{u}$$

For 2-D & 3D problems:

$$\text{div } \underline{\underline{\sigma}} + \underline{\underline{b}} = f \ddot{\underline{\underline{u}}}$$

Weak form in 3D

MWR (W)



$$G(\underline{u}, \underline{\bar{u}}) \equiv - \int_{\Omega} \underline{\bar{u}} \cdot (\text{div } \underline{\tilde{S}} + \underline{b}) d\Omega - \int_{\Gamma_N} \underline{\bar{u}}_i (t_{(n)i} - S_{ij} n_j) d\Gamma$$

$$= - \int_{\Omega} \underline{\bar{u}}_i [(S_{ij,j}) + b_i] d\Omega - \int_{\Gamma_N} \underline{\bar{u}}_i (t_{(n)i} - S_{ij} n_j) d\Gamma$$

(Product Rule)

Note:  $(\bar{u}_i S_{ij})_{,j} = \bar{u}_{i,j} S_{ij} + \bar{u}_i S_{ij,j}$   
 $-\bar{u}_i S_{ij,j} = -(\bar{u}_i S_{ij})_{,j} + \bar{u}_{i,j} S_{ij}$

$$\Rightarrow G(\underline{u}, \underline{\bar{u}}) = \int_{\Omega} \bar{u}_{ij} S_{ij} d\Omega - \int_{\Omega} \bar{u}_i b_i d\Omega - \int_{\Omega} (\bar{u}_i S_{ij})_{,j} d\Omega - \int_{\Gamma_N} \bar{u}_i (t_{(n)i} - S_{ij} n_j) d\Gamma$$

Divergence Theorem:  $-\int_{\Omega} \text{div}(\underline{\tilde{S}}^T \underline{\bar{u}}) d\Omega = -\int_{\Gamma} (\underline{\tilde{S}}^T \underline{\bar{u}}) \cdot \underline{n} d\Gamma$

$$G(\underline{u}, \underline{\bar{u}}) = \int_{\Omega} \underbrace{\bar{u}_{ij} S_{ij}}_{\bar{\underline{\epsilon}} : \underline{\tilde{S}}} d\Omega - \int_{\Omega} \underbrace{\bar{u}_i b_i}_{\underline{\bar{u}} \cdot \underline{b}} d\Omega - \int_{\Gamma} (\bar{u}_i S_{ij}) n_j d\Gamma - \int_{\Gamma_N} \bar{u}_i (t_{(n)i} - S_{ij} n_j) d\Gamma$$

$t_{(n)} = \underline{\tilde{S}} \underline{n}$

Note:

$$\bar{u}_{ij} = \underbrace{\frac{1}{2}(\bar{u}_{i,j} + \bar{u}_{j,i})}_{\text{sym } \nabla \underline{\bar{u}} (= \bar{\underline{\epsilon}})} + \underbrace{\frac{1}{2}(\bar{u}_{i,j} - \bar{u}_{j,i})}_{\text{skew } \nabla \underline{\bar{u}}}$$

$$-\int_{\Gamma_B} \bar{u}_i t_{(n)i} d\Gamma \quad - \int_{\Gamma_N} \bar{u}_i S_{ij} n_j d\Gamma$$

(HDBC)

$$\bar{u}_{ij} n_j = \bar{\underline{\epsilon}}_{ij} S_{ij} + \frac{1}{2}(\bar{u}_{i,j} - \bar{u}_{j,i}) S_{ij} = \bar{\underline{\epsilon}} : \underline{\tilde{S}} + \text{skew}(\nabla \underline{\bar{u}}) : \underline{\tilde{S}}$$

$$\Rightarrow G(\underline{u}, \underline{u}) = \int_{\Omega} \bar{\underline{\epsilon}} : \underline{\tilde{S}} d\Omega - \int_{\Omega} \underline{\bar{u}} \cdot \underline{b} d\Omega - \int_{\Gamma_N} \underline{\bar{u}} \cdot \underline{t}_{(n)} d\Gamma \quad (\text{PVW}) \text{ (W)}$$

$W_I$                        $-W_E$

Energy form in 3D

One can show that the Vainberg's Theorem is satisfied for the above weak form.

$$\Rightarrow \Pi(\underline{u}) = \int_0^1 G(t\underline{u}, \underline{u}) dt$$

$$= \int_0^1 \left[ \int_{\Omega} \underline{\tilde{\epsilon}} : \underline{\tilde{C}}(t\underline{\tilde{\epsilon}}) d\Omega - \int_{\Omega} \underline{u} \cdot \underline{b} d\Omega - \int_{\Gamma_N} \underline{u} \cdot \underline{t}_{(n)} d\Gamma \right] dt$$

$$\Rightarrow \Pi(\underline{u}) = \int_{\Omega} \frac{1}{2} \underline{\tilde{\epsilon}} : \underline{\tilde{C}} \underline{\tilde{\epsilon}} d\Omega - \int_{\Omega} \underline{u} \cdot \underline{b} d\Omega - \int_{\Gamma_N} \underline{u} \cdot \underline{t}_{(n)} d\Gamma$$